

## Water Cooled Chillers



### AquaForce® 30HX/XW Screw

- 75 to 400 Tons
- HFC-134a - HCAI/OSHPD
- Small Footprint
- Heat Recovery



### AquaEdge® 23XRV

- 175 to 550 Tons
- R-134a, R-513a
- High Efficiency Variable Speed Screw Compressor
- HCAI/OSHPD
- IPLV to .299 KW/ton



### AquaEdge® 19XRV

- 200 to 3,400 Tons
- R-134a, R-513a
- Variable-Speed Centrifugal
- IPLV to .31 kW/Ton
- HCAI/OSHPD



### AquaEdge® 19DV

- 200 to 1000 Tons
- Oil Free Ceramic Bearings
- R-1233zd(E) Refrigerant
- 2-Stage Compressor with VFD
- Free Cooling & Heat Recovery
- Low GWP Refrigerant
- IPLV to .29 kw/ton



### AquaEdge® 19MV

- 300 to 700 Tons
- Oil Free Magnetic Bearings
- R-134a, R-513a Refrigerant
- 2-Stage Compressor with VFD
- Compact Design
- Low GWP Refrigerant
- IPLV to .29 kw/ton

## Air Cooled / Modular Chillers



### AquaForce® 30XV

- 140 to 500 Tons, Nominal
- Variable Speed Compressor and condenser fans
- Greenspeed® Intelligence
- R-134a and R-513a
- Flexible Footprint
- IPLV to 21 IEER
- HCAI/OSHPD



### AquaSnap® 30MP Scroll

- 15 to 71 Tons HCAI/OSHPD
- Connect Modules to 600 Tons
- Heat Recovery Option
- Water or Remote Air-cooled



### AquaSnap® 30RAP Scroll

- 10 to 150 Tons HCAI/OSHPD
- Digital Scroll Compressor
- Up to 16.8 EER
- Variable Speed Condenser Fans
- Factory Hydronic Pump Package Option



### AquaSnap® 30RB Scroll

- 60 to 300 Tons
- Heat Recovery Desuperheater
- Up to 17.1 IEER
- Variable Speed Condenser Fans
- Factory Hydronic Pump Package Option
- HCAI/OSHPD

## Packaged Rooftop Units



### WeatherExpert™ 48LC/50LC

- 6 to 23 Tons
- Up to 21 IEER
- Multi-Zone VAV



### WeatherMaster™/Maker™ 48/50TC(Q)/HC(Q)

- 6 to 25 tons
- Up to 20 Tons Heat Pump



### WeatherMaster™/Maker™/Expert™ 48/50 A, P Series

- 20 to 100 Tons
- VAV, CV, SAV
- Evaporative Condenser Options



### WeatherMaster™/Maker™w/EcoBlue™ 48/50FC(Q), GC(Q), JC

- 3 to 25 Tons (Heat Pump)
- Vane Axial ECM Motor
- Up to 16 SEER HCAI/OSHPD
- ULN Ultra Low Nox

## Classroom Indoor Package Unit



- 2 to 5 Tons Heat Pump
- Wall mounted
- Ultra low sound Level
- Inverter Compressors

## Air-Cooled Condensing Units



### Gemini® Condensing Units

- 6 to 130 Tons HCAI/OSHPD
- Small Footprint
- Single & Dual Circuit
- Digital Compressor

## Variable Refrigerant Flow



### 3-Pipe Heat Recovery/Heat Pump

- 3 to 38 Tons
- Inverter Twin Rotary Compressor
- Turndown to 3,500 BTUH
- Flow Selector Powered via FCU
- Single Phase HR to 12 Tons
- Rooftop FCU 3-5 ton



- Pumps
- Heat Exchangers
- Booster Systems
- \*NA in San Diego



- Plate and Frame Heat Exchangers
- Adiabatic Hybrid Coolers



### 2-Pipe Heat Recovery/Heat Pump

- 3 to 36 Tons HCAI/OSHPD
- Inverter Twin Scroll Compressor
- Single Point Piping & Wiring on all Condensing Unit Sizes



### HVLS Fans

- Direct Drive
- Lightweight
- BACnet



- Roof pipe/duct supports
- Service walkways, ramps, crossovers
- Zero penetration support



- Cooling Towers
- HDPE
- Anti Microbial
- Reduced Weight
- Made in USA

## 100% Outside Air & Energy Recovery



### 100% OA Units

- 3 to 60 Tons
- SAT or RH/T Control
- Heat Pump / Gas Heat
- Packaged and Split



### Energy Recovery Ventilators

- Indoor & Outdoor Installation
- 40 to 13,200 CFM
- Demand Controlled Ventilation
- ECM Fan Motors

## Water-Source Heat Pumps & Indoor Self-Contained



### AquaZone® WSHP

- .5 to 30 Tons
- Vertical/Horizontal
- EER up to 37
- ECM Motors



### OmniZone® SCU

- 5 to 60 Tons
- CAV/VAV
- Air Cooled, Remote Air Cooled, Water Cooled

## Fan Coil Units & Coils



### AirStream® 42

- 200 to 4,000 CFM
- ECM Motors
- 2 & 4 Pipe, DX
- Stack, Cassettes



### Coils

- CHW/HW, DX
- Distributed Steam
- Copper, Stainless Steel, Aluminum

## Packaged Central Plants & Controls



### Modular & Custom CHW/HW Plants

- Space Saving
- Integrated Controls
- Single Point Responsibility
- Reduce Project Costs



### Building Automation/i-Vu

- Factory Engineered & Optimized Programs
- Seamless Integration
- Open Protocol
- Single Point Responsibility

## Air Handling Units



### Fully Custom

- 200 to 300,000+CFM
- Fan Array
- Indirect/Direct Evapor. Cooling
- Energy Recovery
- Made in the USA
- HCAI/OSHPD

### Modular and Semi-Custom

- 400 to 60,500 CFM
- Direct Expansion, CHW Cooling
- Hot Water, Gas Heat
- Energy Recovery + Multi-Zone

## Mixed-Air Equations

Mixed-air temperature

$$T_{MA} = \frac{(T_{OA} \times CFM_{OA}) + T_{RA} \times CFM_{RA}}{CFM_{MA}}$$

Mixed-air enthalpy

$$h_{MA} = \frac{(h_{OA} \times CFM_{OA}) + h_{RA} \times CFM_{RA}}{CFM_{MA}}$$

where h = enthalpy, Btu/lb of dry air

*Adapted from 2017 ASHRAE Handbook - Fundamentals, Ch. 16, Eq. 2.*

## Psychrometric Equations

Sensible load (Btu/h)

$$q_s = 1.10 \times CFM \times (T_2 - T_1)$$

Latent load (Btu/h)

$$q_L = 4840 \times CFM \times (W_2 - W_1)$$

where W = humidity ratio

$$q_L = 0.68 \times CFM \times \text{Delta grains}$$

7000 grains = 1 lb<sub>H2O</sub>

Total load (Btu/h)

$$q_T = 4.5 \times CFM \times (h_2 - h_1)$$

where h = enthalpy, Btu/lb of dry air

Humidification (lb/h)

$$\text{lb/h} = \frac{(CFM \times 60 \times (W_2 - W_1))}{\text{specific volume of air}}$$

where W = humidity ratio

Hydronic load (Btu/h)

$$q_{\text{water}} = 500 \times GPM \times (T_2 - T_1)$$

*Adapted from 2017 ASHRAE Handbook - Fundamentals, Ch. 17, Eq. 1-3; Don Brandt, Fundamentals of Psychrometrics, 2nd ed. (I-P), A Course Book for Self Directed or Group Learning (Atlanta: ASHRAE, 2016); 2020 ASHRAE Handbook - HVAC Systems and Equipment, Ch. 13, Eq 9; HVAC Design Essentials ALL course, Level 1, Section 6 and 7.*

## Envelope Equations

$$Q = U \times A \times (T_2 - T_1)$$

where

Q = heat transfer, Btu/h

U = thermal transmittance, Btu/h·ft<sup>2</sup>·°F

A = area, ft<sup>2</sup>

T<sub>2</sub> = temperature outside, °F

T<sub>1</sub> = temperature inside, °F

SC to SHGC

$$\text{SHGC} = \text{SC} \times 0.864$$

U-factor to R-value

$$1/R = \text{U-factor}$$

Adding R-values

$$R_{\text{total}} = R_1 + R_2 + R_3 + R_4$$

where R = resistance to heat flow, ft<sup>2</sup>·°F·h/Btu

*Adapted from 2017 ASHRAE Handbook - Fundamentals, Ch. 15, Eq. 1 and Ch. 25, Eqs. 7 and 11.*

## Fan Affinity Laws

$$CFM_2 = CFM_1 \times \left( \frac{RPM_2}{RPM_1} \right)$$

$$SP_2 = SP_1 \times \left( \frac{RPM_2}{RPM_1} \right)^2$$

$$HP_2 = HP_1 \times \left( \frac{RPM_2}{RPM_1} \right)^3$$

*Adapted from 2020 ASHRAE Handbook - HVAC Systems and Equipment, Ch. 21, Table 2, Eq. 1.*

## Equipment Efficiencies

Fan efficiency

Note: TSP units are in w.g.

$$\text{Fan efficiency} = \frac{CFM \times TSP}{\text{fan BPH} \times 6356}$$

Pump efficiency

Note: Pump head units are ft w.g.

$$\text{Pump efficiency} = \frac{GPM \times \text{pump head} \times \text{specific gravity [1.0]}}{3960 \times \text{pump BHP}}$$

COP

$$= (\text{EER})/3.412 = 12/(\text{kW/ton})/3.412$$

$$= \text{Chiller capacity/compressor power input}$$

$$= Q_{\text{chiller}} / W_{\text{net,in}}$$

where

$$Q_{\text{chiller}} = \text{chiller load}$$

$$W_{\text{net,in}} = \text{work in}$$

EER

$$= 12/(\text{kW/ton}) = \text{COP} \times 3.412$$

kW/ton

$$= 12/\text{EER} = 12/(\text{COP} \times 3.412)$$

*Adapted from AMCA 205-19 per 2020 ASHRAE Handbook - HVAC Systems and Equipment, Ch. 21; 2020 ASHRAE Handbook - HVAC Systems and Equipment, Ch. 44, Eq. 5 and Ch. 48; AHRI Standard 550/590 (I-P), 2020 Standard for performance Rating of Water-chilling and Heat Pump Water-heating Packages Using the Vapor Compression Cycle (Arlington, VA: Air-Conditioning, Heating, and Refrigeration Institute, 2020); 2017 ASHRAE Handbook - Fundamentals, Ch. 2, Eqs. 14-16*

## Cooling Tower Equations

$$\text{Evaporation GPM} = \text{flow GPM} \times \text{range (°F)} \times 0.001$$

$$\text{Bleed rate GPM} = \frac{\text{evaporation GPM}}{\text{cycles of concentration} - 1}$$

*Adapted from 2020 ASHRAE Handbook - HVAC Systems and Equipment, Ch. 40; 2019 ASHRAE Handbook - HVAC Applications, Ch. 50, Section 2.1, Retention Time.*

## Pump Affinity Laws

$$\frac{GPM_1}{GPM_2} = \frac{RPM_1}{RPM_2} \rightarrow GPM_2 = GPM_1 \left( \frac{RPM_2}{RPM_1} \right)$$

$$\frac{TDH_1}{TDH_2} = \left( \frac{RPM_1}{RPM_2} \right)^2 \rightarrow TDH_2 = TDH_1 \left( \frac{RPM_2}{RPM_1} \right)^2$$

$$\frac{BHP_1}{BHP_2} = \left( \frac{RPM_1}{RPM_2} \right)^3 \rightarrow BHP_2 = BHP_1 \left( \frac{RPM_2}{RPM_1} \right)^3$$

*Adapted from 2020 ASHRAE Handbook - HVAC Systems and Equipment, Ch. 44, Table 1.*

## Air Changes per Hour

$$\text{ACH} = (CFM \times 60)/(\text{area} \times \text{height})$$

$$\text{Convert to CFM from ACH: } CFM = (\text{area} \times \text{height}) \times \text{ACH}/60$$

## Common Conversions

watts to Btu/h Btu/h = 3.412 x watts

HP to Btu/h Btu/h = 2545 x HP

HP to watts watts = 746 x HP

PSI PSI = 2.31 x ft head

tons to Btu/h 1ton = 12,000 Btu/h

MBH to Btu/h 1 MBH = 1,000 Btu/h

BHP to kW 1 BHP = 0.746

Atm to PSI 1 Atm = 14.7 psi

ft<sup>3</sup> to gal 1 ft<sup>3</sup> = 7.5 gal

Steam condensate lb/h - gpm 1000 lb/h condensate = 2 gpm

*Adapted from 2017 ASHRAE Handbook - Fundamentals, Ch. 39.*